# Pipe Wall Thickness Calculation: An Alternative Approach in the Petroleum Industry



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alculations on pipe wall thickness are performed to verify the pipe wall thickness of a selected pipe material specification in offshore/onshore oil and gas projects and are usually part of any oil and gas related piping engineering deliverables.

In most parts of the world, such calculations are generally performed based on ASME B31.3. However, in recentive ars, some shortfalls in this method of calculation have been highlighted. The most common solution proposed was to use a more fit purpose equation against the conservative equation.

### DESIGN

Pipe wall thickness is a function of allowable hoopstress. It is derived from cylindrical stress under mechanics of material. A cylindrical stress has these patterns: Circumferential stress, hoop stress, axial/longitudinal stress and radial stress. In relation to internal pressure, we will use the hoop stress to calculate the pipe wall thickness for thin-walled cylinders (wall thickness less than 1/10th of the radius).

### HOOP STRESS FOR THIN WALL PIPE

The hoop stress can be expressed as:  $\sigma_n = p d/2t$ 

Where

on= hoop stress (MPa, psi)

p = internal pressure in the tube or cylinder (MPa, psi)

d = internal diameter of tube or cylinder (mm, in)

t = tube or cylinder wall thickness (mm, in)

For thick walled cylinders, the Lame's equation will be used to determine pipe wall thickness. This equation is best suited for thicker walled high pressure piping, with wall thickness outside the schedule range (r/t ratio of less than

#### HOOP STRESS FOR THICK WALLPIPE

The hoop stress can be expressed as:

 $\sigma_{c} = ((p_{1} r_{1}^{2} - p_{0} r_{0}^{2}) / (r_{0}^{2} - r_{1}^{2})) - (r_{1}^{2} r_{0}^{2} (p_{0} - p_{1}) / (r_{1}^{2} (r_{0}^{2} - r_{1}^{2})))$ Where

 $\sigma_c = stress$  in circumferential direction (MPa psi)

 $p_l = internal pressure in the tube or cylinder (MPa, psi)$ 

 $p_0 = \text{external pressure in the tube or cylinder (MPa, psi)}$ 

 $\eta = internal radius of tube or cylinder (mm. in)$ 

 $r_0$  = external radius of tube or cylinder (mm, in)

 $r = radius to point in tube or cylinder wall (mm. in) (<math>r_1 < r < r_0$ )

Maximum stress when  $r = r_i$  (inside pipe or cylinder)

Generally, pipe wall thickness is calculated by

- Determining the applicable ASME/ANSI B31 Code.
- Calculating the required thickness for internal pressure.
- Checking the calculated thickness to determine its acceptability for external pressure and other applied loads, if applicable.
- Increasing the calculated thickness, as needed, to account for corrosion allowance and mill tolerance.
- Selecting athickness from an ANSI/API table of standard pipe thickness requirements.

All piping component wall thicknesses specified for the individual piping classes are calculated according to the formulae specified in Para 304.1.2 (3a) of ASME B31.3. and are based on continuous long-term internal design pressure and design temperature, specified corrosion/ erosion allowance (under-tolerance), the manufacturing minus/under tolerance and the threading allowance as applicable.

Piping subject to any additional loading such as shortterm upset conditions, occasional loads, external pressure, thermal loading, live loads, marine motions or bending allowances shall be individually assessed to ensure full compliance with ASME B31.3.

A typical pine class sheet (diagram)

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Material Group	Rating				
Carbon Steel	CL150				
Duplex Stainless Steel 22Cr	CL150				
Duplex Stainless Steel 25Cr	CL150				
Austenitic Stainless Steel 316	CL150				
Austenitic Stainless Steel 316	CL150				
Austenitic Stainless Steel 6Mo	CL150				
Austenitic Stainless Steel 6Mo	CL150				
Titanium	CL150				
Titanium	CL150				
	Carbon Steel  Duplex Stainless Steel 22Cr  Duplex Stainless Steel 25Cr  Austenitic Stainless Steel 316  Austenitic Stainless Steel 316  Austenitic Stainless Steel 6Mo  Austenitic Stainless Steel 6Mo  Titanium				

The following pipe class sheet has higher pressure rating than tabulated in ASME 816.5/ASME 816.34 (diagram )

SI. No.	Material Group	Rating		Piessure, Barg	
			Tem p. °C	Pipe Class Sheet	AS ME 816.5
	Duplex	CL300	100	51.5	50.7
1.	Stainless		150	50.2	45.9
	Steel 22Cr		200	48.6	42.7
2.	Austenitic Stainless	CL300	100	51.5	50.7
	Steel 6Mo		150	50.2	50.2
	Duplex Stainless	CL600	100	10.3.0	101.3
3.	Steel 22Cr		150	100.3	91.9
			200	97.2	85.3
			250	92.7	80.9
4.	Duplex Stainless Steel 22 Cr	CL600	100	10.3.0	101.3
5.	Duplex Stainless Steel 25Cr	CL600	100	10.3.0	101.3
6.	Austenitic Stainless	CL600	100	103.0	101.3
	Steel 6Mo		150	100.3	91.9
7.	Duplex Stainless Steel 22Cr	CL900	150	140.0	137.8

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SI. No.	Material Group	Rating	Temp.	Pressure, Barg	
			°C	Pipe Class Sheet	ASME B16.5
8.	Duplex Stainless Steel 22Cr	CL900	100	154.6	152.0
	Austenitic Stainless	CL900	100	154.6	152.0
9.	Steel 6Mo		150	150.5	137.8
			200	131.4	128.0
10	Duplex Stainless	CL 1500	100	257.6	253.3
10.	Steel 22Cr		150	250.8	229.6
	Austenitic Stainless	CL 1500	100	257.6	253.3
	Steel 6Mo		150	250.8	229.6
11.			200	219.0	213.3
			250	209.7	202.3
			300	199.1	194.3
12	Duplex Stainless Steel 22Cr	CL2500	150	395.0	382.7
13.	Duplex Stainless Steel 25Cr	CL2500	150	394.1	382.7
14.	Austenitic Stainless	CL2500	100	429.4	422.2
	Steel 6Mo		150	418.1	382.7
15.	Duplex Stainless Steel 22Cr	CL4500	140	715.3	703.1

As individually noted in the following pipe class sheet, for certain sizes the selected pipe wall thickness is lower than the calculated value. This is because the Lame's equation used gives a lower wall thickness where the difference is small.

\$I. No.	Size, NPS	Material Group	Rating	Calculated thickness, mm based on hoop stress	Calculated thickness, mm based on Lame's equation	Difference, mm
1.	12	Austenitic Stainless Steel 6Mo	CL900	14.35	14.27	(-)0.08
2.	16	Duplex Stainless Steel 22Cr	CL1500	16.76	16.66	(-) 0.10
3.	4	Duplex Stainless Steel 22Cr	CL1500	6.06	6.02	(-) 0.04
4.	24	Austenitic Stainless Steel 6Mo	CL1500	38.93	38.89	(-) 0.04
5.	1.5	Duplex Stainless	CL2500	5.10	5.08	(-) 0.02
٥.	10	Steel 22Cr		28.85	28.58	(-) 0.27
	8	Duplex Stainless	CL2500	18.37	18.26	(-) 0.11
Ó.	12	Steel 25Cr		25.54	25.40	(-) 0.14
	14			28.01	27.79	(-) 0.22
	6	Duplex Stainless	CL4500	25.23	25.00	(-) 0.23
	10	Steel 22Cr		40.08	40.00	(-) 0.08
	12			40.47	40.00	(-) 0.47
8.	8	Duplex Stainless Steel 22Cr	CL4500	36.11	36.00	(-) 0.11
9.	0.5	Austenitic Stainless	CL4500	2.79	2.77	(-) 0.02
	6	Steel 6Mo		22.08	21.95	(-) 0.13



## THE CHALLENGE

The main challenge identified in pipe wall thickness calculation is the alternative formula against the conservative formulae. The reason for the difference in pipe wall thickness calculations is based on a different formula used in the calculation. ASME B31.3 gives a more conservative calculation which, in general, is more accurate for thinner walled scheduled piping with thickness below (t<D/b).

The alternative formula is based on the Lame's equation where variables such as external pressure are given more consideration in the calculation. Lame's equation is better used for thicker walled high pressure piping, with wall thicknesses outside the schedule range (t>D/6 or P/SE >0.385).

Section 304.1.2 (a), ASME B31.3 formula with t < D/6 and P/SE<0.385 is used for validation. The conditions/criteria and application of the formula, as given in section 304.1.2 (a), meet the design requirement. Hence, the design is accordance with section 304.1.2 (a), ASME B31.3 and the provisions of section 304.1.2 (b) should not be used.

However, under certain circumstances (refer to reasons mentioned below), the most common solution proposed will be the acceptance of using a fit purpose equation against the conservative equation.

- Pipe wall thickness calculations aren't part of the discipline engineering deliverables.
- Difference of the adjoulated wall thickness is negligible and most of the sizes are not used in a particular project.

 Most of the line which uses the affected pipe classes are not designed as per maximum design, where the pipe wall thickness is calculated based on the maximum design pressure and temperature.

### CONCLUSION

Pipe wall thickness calculation shall be performed based on ASME B31.3. However, issues such as acceptance of using a more fit equation can be accepted as the option for an alternative method.

# Author's Bio data